

A 3D architectural rendering of a stadium, likely the Stade de France, with a semi-transparent mesh overlaid on the seating areas. The mesh is colored in shades of orange and red, indicating the computational domain for the CFD study. The stadium is shown from an elevated perspective, showing the pitch, stands, and surrounding structures.

CFD study of PM10 dispersion in a sports stadium using a mesh based on a geometry obtained from a 3-D cloud of laser points

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1 Introduction

2 Methodology

3 Mesh generation strategy

4 Numerical framework

5 Numerical results

6 Conclusion

Context and motivation

Context

- **Indoor air quality** and **thermal comfort** → important stakes in large structures such as sport facilities
- Such elements are dependent on the heating, ventilation, and air conditioning systems (HVAC)
- Numerical simulations can provide insightful data to optimize/design HVAC

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Motivation

For large sports facilities :

- Computational Fluid dynamics can provide useful/complementary local data
but ...
- CFD prediction accuracy for complex thermo-aeraulic systems ?
- Reproduction of such detailed geometry ?

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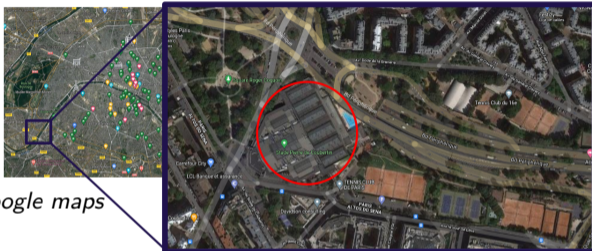
Proposition

One possible methodology for complex geometries CFD simulation using a high fidelity mesh based on 3D cloud of points

Methodology

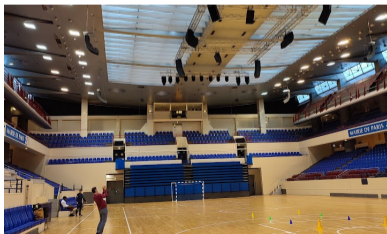
Pierre de Coubertin stadium

Used for the Paralympic and Olympic Games



Around **4.5 k** seats

© Google maps

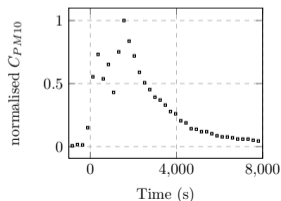


Methodology

Test case introduction

Reproduction using CFD of Particle Matter (PM10) during a handball game within the enclosed stadium

- Experimental campaign done by different laboratories during summer 2021 (*CSTB, LISA, LCPP, CERE*A)
- Local peak of PM10 concentration C_{PM10} caused by fireworks **in** and **outside** the stadium
- Use of the open source CFD software code *saturne*
- Extension of the validation of a novel indoor air flow algorithm
[Amino, H. Development of a CFD time scheme for indoor airflow applications, PhD, 2022]

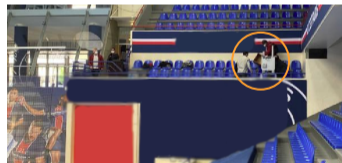


Difficulties

- Very detailed system
- Lack of information regarding the stadium heterogeneous fields



Fireworks outside the stadium



PM10 Sensor location (CSTB, LISA, LCPP, CERE)A)

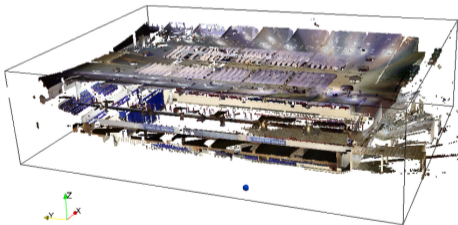
PM10 concentration measured over time
(normalised values)

Mesh generation strategy

Presentation of 3D cloud of points based mesh : main steps

1 Cloud of points generation and pre-processing

- Acquisition using a 3-D laser scanner
- Parasite points cleaning
- Rotation and translation into the the reference axis



Stadium cloud of points [Lefranc Y., 2021]



Cloud of points inside visualization

Mesh generation strategy

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2 Fluid volume definition

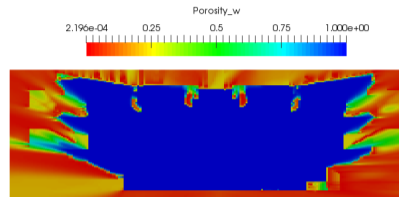
- Homogeneous box domain definition incorporating the cloud of points
- Transport of the porosity $\Pi \in [0, 1]$ with source terms defined at the scanner locations

$$\frac{\partial \Pi}{\partial t} + \text{div}(\Pi \underline{e}_r) - \text{div}(\underline{e}_r) = 0$$

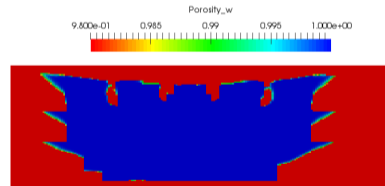
- Fluid volume if porosity \geq threshold (variable in space)



Porosity thresholds used to create the mesh



Porosity field for a threshold of 2.2×10^{-4} : fake fluid cells



Porosity field for a threshold of 0.98 : lack of details

← compromise !

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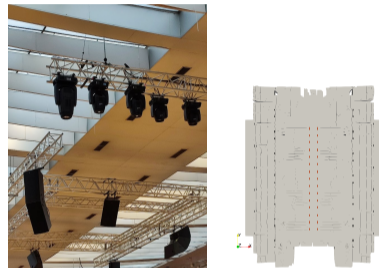
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- Fluid volume if porosity \geq **threshold** (variable in space)

3 Boundary zones definition



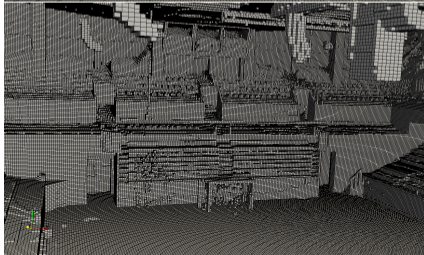
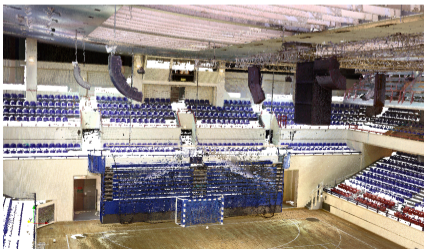
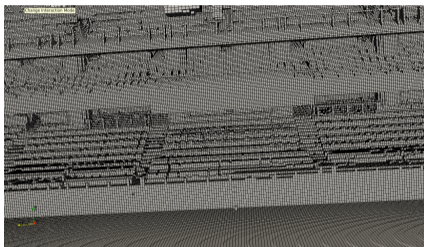
(Left) Illustration of the extraction vents. (Right) Corresponding zones.



Inside visualization of the outlets zones (in red)

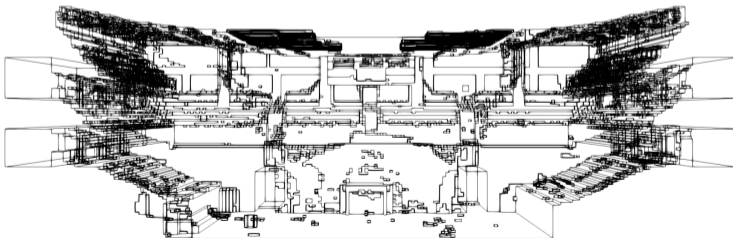
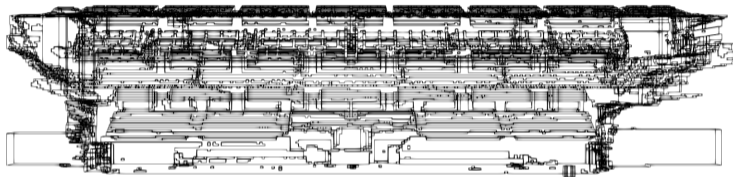
Mesh used for simulations

Final mesh composed of $2.4 \cdot 10^6$ hexahedral cells ($\Delta x = \Delta y = \Delta z = 0.2\text{m}$)



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Final mesh composed of $2.4 \cdot 10^6$ hexahedral cells ($\Delta x = \Delta y = \Delta z = 0.2\text{m}$)

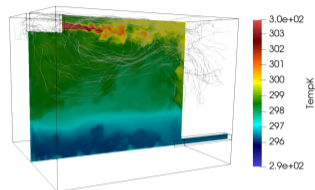


Different views of the mesh retained for the simulations

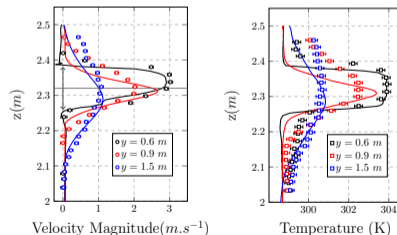
Numerical choices

Indoor airflow time scheme [Amino et. al., IJNMF, 2022]

- Navier–Stokes equations
- Implemented in `code_saturne` (F.V method)
- Prediction correction sub-iterative scheme
- Theta scheme ($\theta = 1$: Euler implicit, $\theta = 1/2$: Crank Nicolson)
- Able to predict incompressible and compressible flow
 - Account for the total pressure variation
 - Total energy conservation
- Validation in the indoor flow framework made [Amino H., Flageul C., Tiselj, I., Benhamadouche S., Carissimo, B., Ferrand M., 2022. A time-staggered second order conservative time scheme for variable density flow, IJNMF]



Validation example : snapshot of a hot ventilation jet



Velocity and temperature vertical profiles for different axial distances

Numerical choices

Simulation setup

- Isothermal ventilation
- $Q_{in} = 60 \cdot 10^3 \text{ m}^3 \cdot \text{h}^{-1}$
($\approx 3 \times \Omega_{tot}/h$)
- $k - \varepsilon$ model for turbulence
- Transient inlet condition :
 - $t \leq 1550 \text{ s}$ (peak) : inlet of PM10 and air
 - else : inlet of pure air

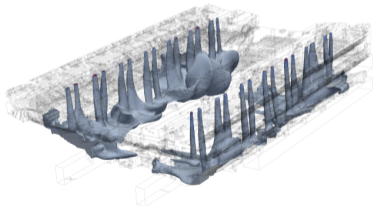
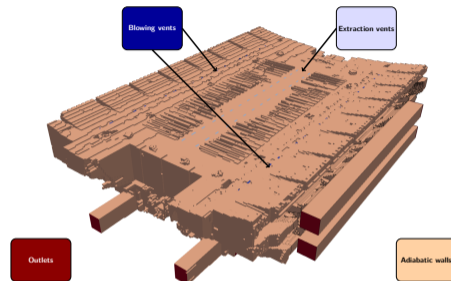
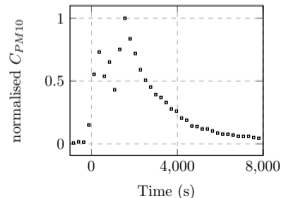


Illustration of the PM10 injection



Boundary conditions zones imposed for this simulation.

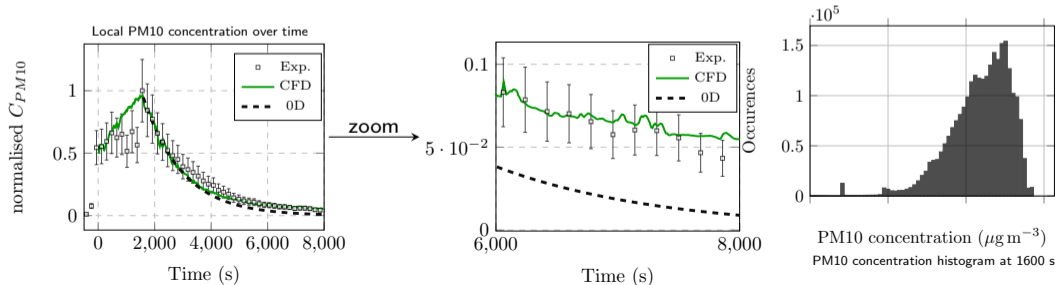


PM10 concentration measured over time.

Results

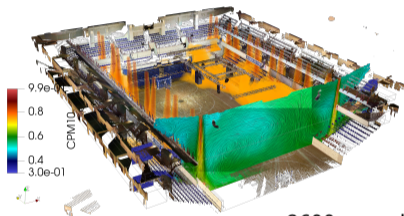
CFD results compared to the experimental data and a simplified 0D model (homogeneous mixture)

- Evolution shape agreement
- Larger error for the simplified model

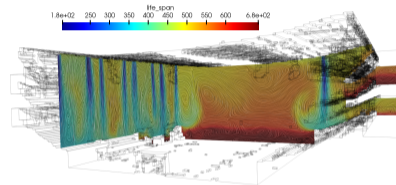


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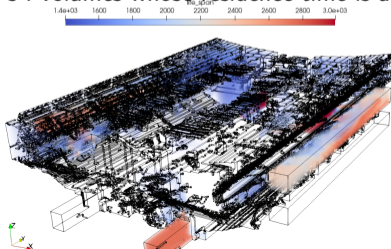
CFD also provides 3D insights of the indoor air flow such as :
the local dynamic field



the local residence time



$t = 3600 \text{ s}$: volumes whose residence time is above 1400 s



Conclusion

- A methodology leading to **high fidelity** meshes is presented and applied to a stadium CFD study
- Numerical results were in agreement with the experimental data
- Such validation result emphasises the methodology potential in the CFD simulation framework
- Extended to external simulations

Further method improvements :

- Use of more advanced immersed boundary methods e.g. :

[Narvaez, et al., 2023, Automatic Solid Reconstruction from 3-D Points Set for Flow Simulation via an Immersed Boundary Method. FVCA]

- Boundary zones identification/automatization

Conclusion

Thank you for your attention.

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Acknowledgements :

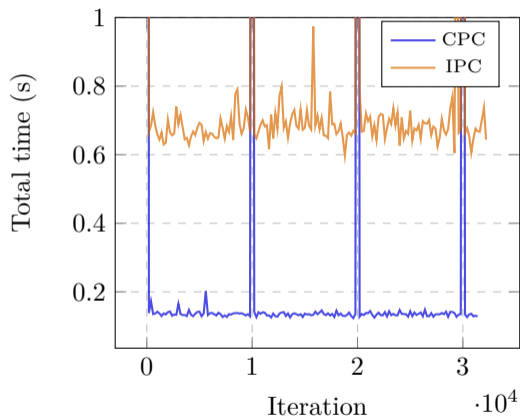
EDF R&D and CEREА : Yannick Lefranc, Aurélien Facheux for the 3D cloud of points measurements

CSTB, LISA, LCPP, CEREА for experimental data and discussions

Mairie de Paris for the visits and informations regarding the stadium

Financial support by ANRT through an EDF-CIFRE contract

Simulation time



Coubertin simulation (100 physical seconds, 2M cells, 175 procs) isothermal problem

Solver	IPC time (s)	CPC time (s)
Temperature	1.17	3.3483
Velocity	10.4	11.8
k	2.38	2.84
ε	2.6	3.0
Pressure	2143.5 (2096.2)	195.1
Total time	2206.4	260.4

Scheme equations

Navier–Stokes compressible equations :

$$\frac{\partial \rho}{\partial t} + \operatorname{div}(\rho \underline{u}) = 0,$$

$$\frac{\partial \rho \underline{u}}{\partial t} + \operatorname{div}(\underline{u} \otimes \rho \underline{u}) = -\nabla p + \operatorname{div}(\underline{\tau}) + \overbrace{\underline{f}}^{\rho \underline{g}},$$

$$\frac{\partial(\rho e)}{\partial t} + \operatorname{div}(e \rho \underline{u}) = -p \operatorname{div}(\underline{u}) + \underline{\tau} : \underline{\nabla} \underline{u} + \operatorname{div}(\lambda \nabla T),$$

$$\frac{\partial(\rho Y)}{\partial t} + \operatorname{div}(Y \rho \underline{u}) = \operatorname{div}(K \nabla Y),$$

$$T = \mathcal{T}(\rho, e) = \frac{\gamma - 1}{R_a} e$$

$$p = \mathcal{P}(\rho, e) = \rho R_a T.$$

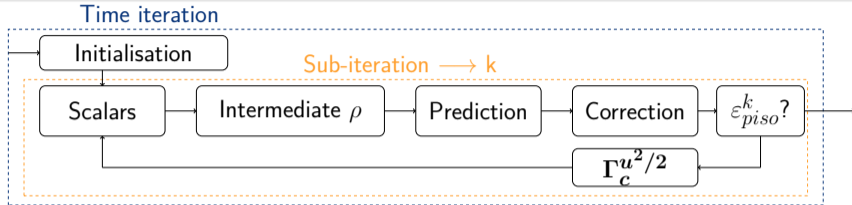
Set of equations chosen to be discretised

Density variation

- Buoyant effects

Local Thermodynamic
pressure variation

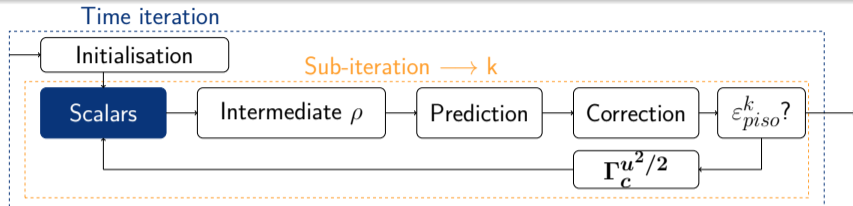
Proposition : compressible pressure correction (CPC) scheme



Features

- Variable time convergence order
- Total energy conservation
- ρ dependence on T
- EOS linearisation
- ρ dependence on p

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Internal energy equation solved in $[n; n + 1, k] \rightarrow e_c^{n+1,k}$

$$\frac{\rho_c^{n+1,k-1} e_c^{n+1,k} - \rho_c^{n,k-1} e_c^n}{\Delta t} + \text{Div}_c \left(\left\langle \Theta \left(e^n, e^{n+1,k} \right) \right\rangle_f \underline{q}_f \Big|_n^{n+1,k-1} \right) = \mu \overbrace{(S_c^2)^{n+\theta,k-1}}^{\text{Dissipation function}} + \Gamma_c u^2/2 \Big|_n^{n+1,k-1}$$

$$+ \text{Lapl}_c \left(\lambda, \Theta \left(T^n, T^{n+1,k} \right) \right) - \text{Div}_c \left(\left\langle \Theta \left(p^n, p^{n+1,k-1} \right) \underline{u}^{n+\theta,k-1} \right\rangle_f \right) + \underline{u}_c^{n+\theta,k-1} \cdot \underline{\nabla}_c p \Big|_{n-1+\theta}^{n+\theta,k-1}$$

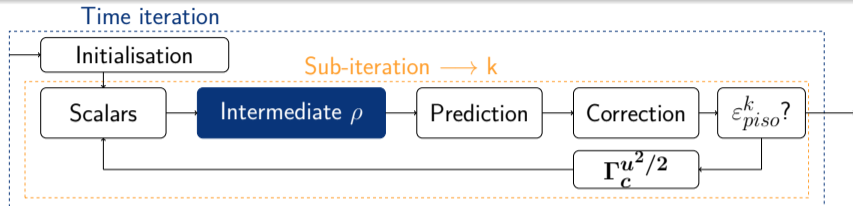
$\Gamma_c u^2/2$: corrective source term \rightarrow kinetic energy dissipation, derived from its discrete equation [Amino et al., 2022] adapted from Herbin et al., 2020

$$T_c^{n+1,k} = e_c^{n+1,k} c_v^{-1}$$

[Amino et al., A time-staggered second order conservative time scheme for variable density flow. IJNMF, 2022]

[Herbin, Zaza, Latché, A cell-centered pressure-correction scheme for the compressible Euler equations, IMA Journal of Numerical Analysis, 40,

Proposition : compressible pressure correction (CPC) scheme



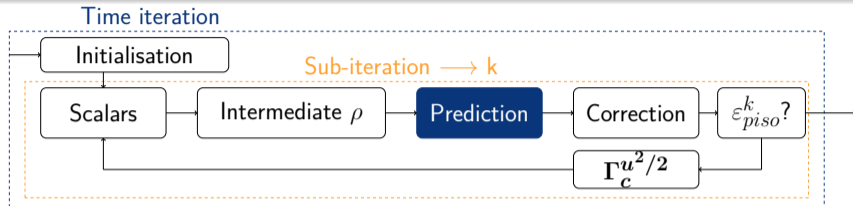
Features

- Variable time convergence order
- Total energy conservation
- ρ dependence on T
- EOS linearisation
- ρ dependence on p

Predicted density $\rightarrow \tilde{\rho}^k$

$$\tilde{\rho}^k = \frac{p^{n+1,k-1}}{R_a T^{n+1,k}}$$

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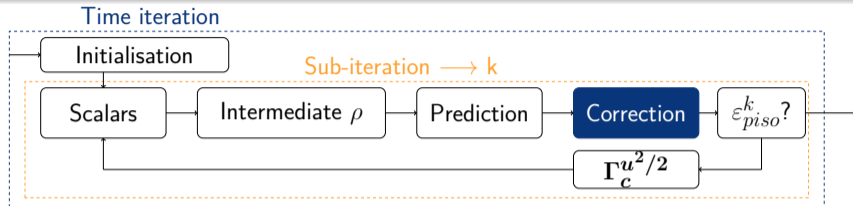
Momentum equation solved between $[n - 1 + \theta; n + \theta, k] \rightarrow \underline{\tilde{u}}^k$

$$\frac{\Theta(\rho_c^n, \rho_c^{n+1, k-1}) \underline{\tilde{u}}^k - \Theta(\rho_c^{n-1}, \rho_c^{n, k-1}) \underline{u}_c^{n-1+\theta}}{\Delta t} + \text{Div}_c \left(\left\langle \Theta(\underline{u}^{n-1+\theta}, \underline{\tilde{u}}^k) \right\rangle_f \otimes \underline{q}_f \Big|_{n-1+\theta}^{n+\theta, k-1} \right)$$

$$= -\text{Grad}_c \left(\left\langle \rho \Big|_{n-1+\theta}^{n+\theta, k-1} \right\rangle_f \right) + \text{Div}_c \left(\left\langle \Theta(\underline{\tau}^n, \underline{\tau}^k) \right\rangle_f \right) + \underline{f}_c \Big|_{n-1+\theta}^{n+\theta, k-1}.$$

$$\underline{\tau} = \mu \left(\underline{\nabla} \underline{u} + \underline{\nabla} \underline{u}^T \right) - \frac{2}{3} \mu \text{div}(\underline{u}) \underline{I} \quad (\text{shear stress tensor})$$

Proposition : compressible pressure correction (CPC) scheme



Features

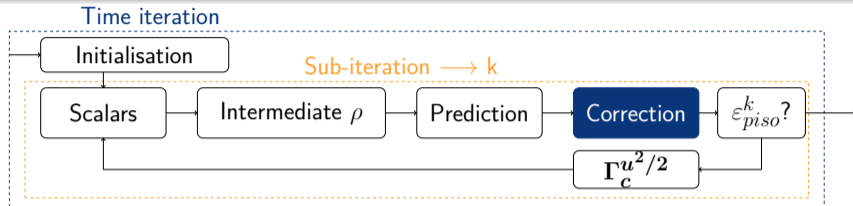
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Mass and simplified momentum equations solved between $[n; n + 1, k]$

$$\left\{ \begin{array}{l} \frac{\Theta(\rho_c^n, \rho_c^{n+1,k}) \underline{u}_c^{n+\theta,k} - \Theta(\rho_c^n, \rho_c^{n+1,k-1}) \tilde{\underline{u}}_c^k}{\Delta t} + \underline{\nabla}_c \phi^k = 0, \\ \text{Div}_c \left(\underline{q}_f \Big|_n^{n+1,k} \right) + \frac{\rho_c^{n+1,k} - \rho_c^n}{\Delta t} = 0, \quad \phi_c^k = \Theta \left(p_c \Big|_{n-2+\theta}^{n-1+\theta}, \rho_c^{n+1,k} \right) - p_c \Big|_{n-1+\theta}^{n+\theta, k-1}. \end{array} \right.$$

$$\rho_c^{n+1,k} = \tilde{\rho}_c^k + \left(p_c^{n+1,k} - p_c^{n+1,k-1} \right) \left(\frac{\partial \rho}{\partial p} \right)_T \left(p_c^{n+1,k-1}, T_c^{n+1,k} \right)$$

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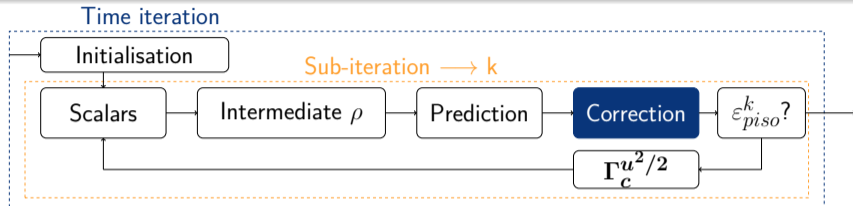
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$$\underline{q}_f \Big|_n^{n+1,k} = \left\langle \Theta(\rho^n, \rho^{n+1,k}) \underline{u}^{n+\theta,k} \right\rangle_f = \left\langle \Theta(\rho^n, \rho^{n+1,k-1}) \tilde{\underline{u}}^k \right\rangle_f - \Delta t \underline{\nabla}_f \phi^k$$

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Features

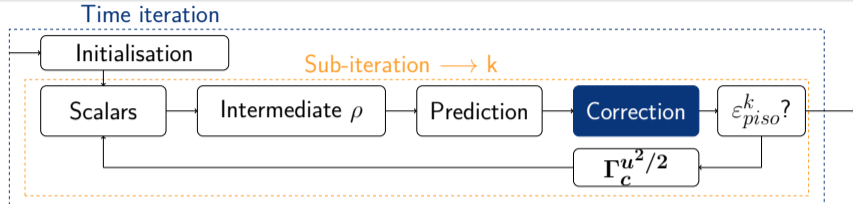
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Helmholtz equation for $p_c^{n+1,k}$:

$$\frac{1}{\Delta t} \left(\frac{\partial \rho}{\partial p} \right)_T (p_c^{n+1,k-1}, T_c^{n+1,k}) p_c^{n+1,k} - \theta \text{Lapl}_c (\Delta t, p_c^{n+1,k}) = - \frac{(\tilde{\rho}_c^k - \rho_c^n)}{\Delta t}$$

$$- \text{Div}_c \left(\left\langle \Theta (\rho^n, \rho^{n+1,k-1}) \tilde{u}^k + \Delta t (\underline{\nabla} p|_{n-1+\theta}^{n+\theta, k-1} + \delta \underline{f}^k) \right\rangle_f \right) + (1 - \theta) \text{Lapl}_c (\Delta t, p|_{n-2+\theta}^{n-1+\theta})$$

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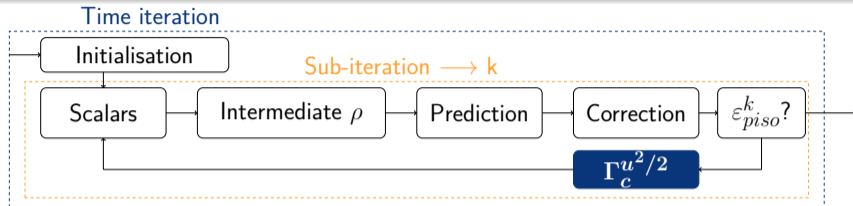
Density update :

$$\rho_c^{n+1,k} = \frac{\rho_c^{n+1,k}}{R_a T_c^{n+1,k}}$$

Velocity update :

$$\Theta(\rho_c^n, \rho_c^{n+1,k}) \underline{u}_c^{n+\theta,k} = \Theta(\rho_c^n, \rho_c^{n+1,k-1}) \tilde{\underline{u}}_c^k - \Delta t \underline{\nabla}_c \phi^k$$

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Based on the discrete kinetic energy equation :

$$\frac{\Theta(\rho_c^n, \rho_c^{n+1,k}) |\underline{u}_c^{n+\theta,k}|^2 - \Theta(\rho_c^{n-1}, \rho_c^{n,k-1}) |\underline{u}_c^{n-1+\theta}|^2}{2\Delta t} + \text{Div}_c \left(\frac{\left| \left\langle \Theta(\underline{u}^{n-1+\theta}, \tilde{\underline{u}}^k) \right\rangle_f \right|^2}{2} \underline{q}_f \Big|_{n-1+\theta}^{n+\theta,k-1} \right)$$

$$= -\Gamma_c^{u^2/2} \Big|_n^{n+1,k} + \overbrace{\Gamma_c^p \Big|_n^{n+1}}^{\text{second order term}} - \text{Grad}_c \left(\left\langle \rho \Big|_{n-1+\theta}^{n+\theta,k} \right\rangle_f \right) \cdot \underline{u}_c^{n+\theta,k}.$$

Adapted from Herbin et al, 2020, Eq. (e) + Eq. $\left(\frac{|u|^2}{2} \right) \rightarrow \text{Eq. } (E_{tot})$